

Multiple-Choice Test
Finite Difference Method
Ordinary Differential Equations
COMPLETE SOLUTION SET

1. The exact solution to the boundary value problem

$$\frac{d^2 y}{dx^2} = 6x - 0.5x^2, \quad y(0) = 0, \quad y(12) = 0$$

for $y(4)$ is

- (A) -234.67
(B) 0.00
(C) 16.000
(D) 37.333

Solution

The correct answer is (A).

First separate the variables.

$$\begin{aligned}\frac{d^2 y}{dx^2} &= 6x - 0.5x^2 \\ \frac{d}{dx} \left(\frac{dy}{dx} \right) &= 6x - 0.5x^2 \\ \frac{dy}{dx} &= \int (6x - 0.5x^2) dx \\ &= \frac{6x^2}{2} - \frac{0.5x^3}{3} + C \\ &= 3x^2 - \frac{0.5}{3}x^3 + C \\ y &= \int \left(3x^2 - \frac{0.5}{3}x^3 + C \right) dx \\ &= \frac{3}{3}x^3 - \frac{0.5}{12}x^4 + Cx + D \\ &= x^3 - \frac{0.5}{12}x^4 + Cx + D\end{aligned}$$

Set $y(0) = 0$ to solve for D .

$$y(0) = 0^3 - \frac{0.5}{12} \times 0^4 + C \times 0 + D$$

$$0 = D$$

Set $y(12) = 0$ to solve for C .

$$y(12) = 12^3 - \frac{0.5}{12} \times 12^4 + C \times 12$$

$$0 = 1728 - 864 + 12C$$

$$-12C = 864$$

$$C = -72$$

Thus,

$$y = x^3 - \frac{0.5}{12} x^4 + Cx + D$$

$$= x^3 - \frac{0.5}{12} x^4 - 72x$$

$$y(4) = 4^3 - \frac{0.5}{12} \times 4^4 - 72 \times 4$$

$$= 64 - 10.6667 - 288$$

$$= -234.67$$

2. Given

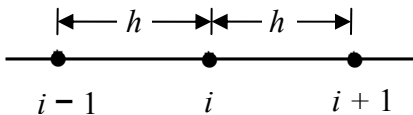
$$\frac{d^2 y}{dx^2} = 6x - 0.5x^2, \quad y(0) = 0, \quad y(12) = 0$$

the value of $\frac{d^2 y}{dx^2}$ at $y(4)$ using the finite difference method and a step size of $h = 4$ can be approximated by

- (A) $\frac{y(8) - y(0)}{8}$
- (B) $\frac{y(8) - 2y(4) + y(0)}{16}$
- (C) $\frac{y(12) - 2y(8) + y(4)}{16}$
- (D) $\frac{y(4) - y(0)}{4}$

Solution

The correct answer is (B).



The finite difference approximation for $\frac{d^2 y}{dx^2}$ at node i , that is $x = x_i$, is given by

$$\left. \frac{d^2 y}{dx^2} \right|_i \approx \frac{y_{i+1} - 2y_i + y_{i-1}}{h^2}$$

where

$$y_{i+1} = y(8)$$

$$y_i = y(4)$$

$$y_{i-1} = y(0)$$

$$h = 4$$

Thus

$$\begin{aligned} \frac{d^2 y}{dx^2} &\approx \frac{y(8) - 2y(4) + y(0)}{(4)^2} \\ &\approx \frac{y(8) - 2y(4) + y(0)}{16} \end{aligned}$$

3. Given

$$\frac{d^2y}{dx^2} = 6x - 0.5x^2, \quad y(0) = 0, \quad y(12) = 0$$

the value of $y(4)$ using the finite difference method with a second order accurate central divided difference method and a step size of $h = 4$ is

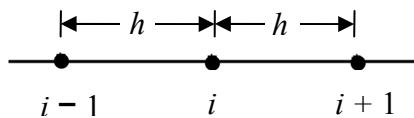
- (A) 0.000
- (B) 37.333
- (C) -234.67
- (D) -256.00

Solution

The correct answer is (D).

$$\frac{d^2y}{dx^2} - 6x - 0.5x^2 = 0$$

now



The finite difference approximation for $\frac{d^2y}{dx^2}$ at node i , that is $x = x_i$, is given by

$$\left. \frac{d^2y}{dx^2} \right|_i \approx \frac{y_{i+1} - 2y_i + y_{i-1}}{h^2}$$

Thus

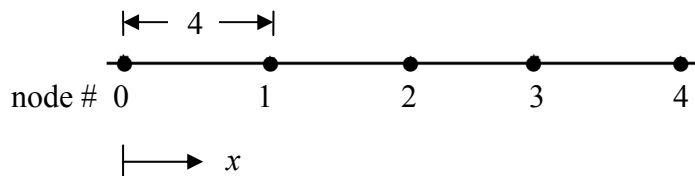
$$\frac{y_{i+1} - 2y_i + y_{i-1}}{(h)^2} - 6x_i + 0.5x_i^2 = 0$$

$$y_{i+1} - 2y_i + y_{i-1} - 6x_i h^2 + 0.5x_i^2 h^2 = 0$$

$$y_{i+1} - 2y_i + y_{i-1} = 6x_i h^2 - 0.5x_i^2 h^2$$

$$= 6x_i 4^2 - 0.5x_i^2 4^2$$

$$= 96x_i - 8x_i^2$$



At node $i = 0, x_0 = 0$

$$y_0 = 0$$

At node $i = 1, x_1 = 4$

$$\begin{aligned} y_2 - 2y_1 + y_0 &= 96x_1 - 8x_1^2 \\ &= 96(4) - 8(16) \\ &= 256 \end{aligned}$$

At node $i = 2, x_2 = 8$

$$\begin{aligned} y_3 - 2y_2 + y_1 &= 96x_2 - 8x_2^2 \\ &= 96(8) - 8(64) \\ &= 256 \end{aligned}$$

At node $i = 3, x_3 = 12$

$$y_3 = 0$$

Writing the above equations in matrix form

$$\begin{bmatrix} 1 & 0 & 0 & 0 \\ 1 & -2 & 1 & 0 \\ 0 & 1 & -2 & 1 \\ 0 & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} y_0 \\ y_1 \\ y_2 \\ y_3 \end{bmatrix} = \begin{bmatrix} 0 \\ 256 \\ 256 \\ 0 \end{bmatrix}$$

which gives

$$y(0) = y_0 = 0$$

$$y(4) \approx y_1 = -256$$

$$y(8) \approx y_2 = -256$$

$$y(12) = y_3 = 0$$

Hence

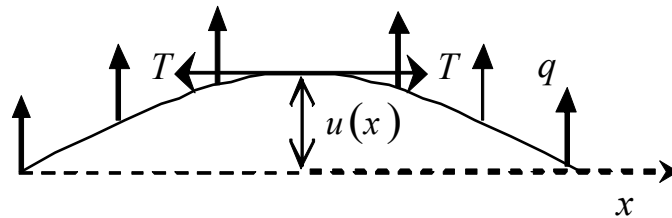
$$y(4) \approx -256$$

4. The transverse deflection u of a cable of length L that is fixed at both ends, is given as a solution to

$$\frac{d^2u}{dx^2} = \frac{Tu}{R} + \frac{qx(x-L)}{2R}$$

where

T = tension in cable
 R = flexural stiffness
 q = distributed transverse load



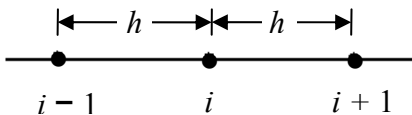
Given $L = 50''$, $T = 2000 \text{ lbs}$, $q = 75 \frac{\text{lbs}}{\text{in}}$, and $R = 75 \times 10^6 \text{ lbs} \cdot \text{in}^2$

Using finite difference method modeling with second order central divided difference accuracy and a step size of $h = 12.5''$, the value of the deflection at the center of the cable most nearly is

- (A) 0.072737"
- (B) 0.080832"
- (C) 0.081380"
- (D) 0.084843"

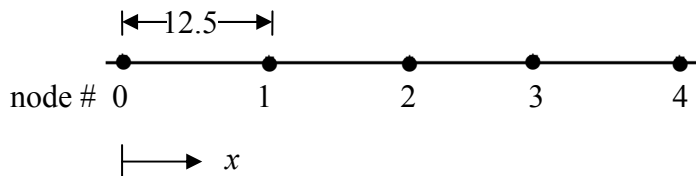
Solution

The correct answer is (D).



At node i , using finite difference approximations,

$$\frac{d^2u}{dx^2} \Big|_i \approx \frac{u_{i+1} - 2u_i + u_{i-1}}{h^2} = \frac{(2000)u_i}{(75 \times 10^6)} + \frac{75 \times x_i(x_i - 50)}{2(75 \times 10^6)}$$



For $i = 0$, $x_0 = 0$

$$u_0 = 0 \quad (1)$$

For $i = 1$, $x_1 = 12.5$

$$\frac{u_2 - 2u_1 + u_0}{h^2} = \frac{(2000)u_1}{(75 \times 10^6)} + \frac{75 \times x_1(x_1 - 50)}{2(75 \times 10^6)}$$

$$\frac{u_2 - 2u_1 + u_0}{12.5^2} = \frac{(2000)u_1}{(75 \times 10^6)} + \frac{75 \times 12.5(12.5 - 50)}{2(75 \times 10^6)}$$

$$\frac{u_2 - 2u_1 + u_0}{156.25} = 2.6667 \times 10^{-5} u_1 - 2.3438 \times 10^{-4}$$

$$u_2 - 2u_1 + u_0 = 4.1667 \times 10^{-3} u_1 - 3.6621 \times 10^{-2}$$

$$u_2 - 2.0042u_1 + u_0 = -3.6621 \times 10^{-2} \quad (2)$$

For $i = 2$, $x_2 = 25$

$$\frac{u_3 - 2u_2 + u_1}{h^2} = \frac{(2000)u_2}{(75 \times 10^6)} + \frac{75 \times x_2(x_2 - 50)}{2(75 \times 10^6)}$$

$$\frac{u_3 - 2u_2 + u_1}{12.5^2} = \frac{(2000)u_2}{(75 \times 10^6)} + \frac{75 \times 25(25 - 50)}{2(75 \times 10^6)}$$

$$\frac{u_3 - 2u_2 + u_1}{156.25} = 2.6667 \times 10^{-5} u_2 - 3.1250 \times 10^{-4}$$

$$u_3 - 2u_2 + u_1 = 4.1667 \times 10^{-3} u_2 - 4.8828 \times 10^{-2}$$

$$u_3 - 2.0042u_2 + u_1 = -4.8828 \times 10^{-2} \quad (3)$$

For $i = 3$, $x_3 = 37.5$

$$\frac{u_4 - 2u_3 + u_2}{h^2} = \frac{(2000)u_3}{(75 \times 10^6)} + \frac{75 \times x_3(x_3 - 50)}{2(75 \times 10^6)}$$

$$\frac{u_4 - 2u_3 + u_2}{12.5^2} = \frac{(2000)u_3}{(75 \times 10^6)} + \frac{75 \times 37.5(37.5 - 50)}{2(75 \times 10^6)}$$

$$\frac{u_4 - 2u_3 + u_2}{156.25} = 2.6667 \times 10^{-5} u_3 - 2.3438 \times 10^{-4}$$

$$u_4 - 2u_3 + u_2 = 4.1667 \times 10^{-3} u_3 - 3.6621 \times 10^{-2}$$

$$u_4 - 2.0042u_3 + u_2 = -3.6621 \times 10^{-2} \quad (4)$$

For $i = 4$, $x_4 = 50$

$$u_4 = 0 \quad (5)$$

Rewrite Equations 1-5 in matrix form

$$\begin{bmatrix} 1 & 0 & 0 & 0 & 0 \\ 1 & -2.0042 & 1 & 0 & 0 \\ 0 & 1 & -2.0042 & 1 & 0 \\ 0 & 0 & 1 & -2.0042 & 1 \\ 0 & 0 & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} u_0 \\ u_1 \\ u_2 \\ u_3 \\ u_4 \end{bmatrix} = \begin{bmatrix} 0 \\ -3.6621 \times 10^{-2} \\ -4.8828 \times 10^{-2} \\ -3.6621 \times 10^{-2} \\ 0 \end{bmatrix}$$

Solving this matrix by using Gauss elimination gives

$$u_0 = 0''$$

$$u_1 = 0.060605''$$

$$u_2 = 0.084843''$$

$$u_3 = 0.060605''$$

$$u_4 = 0''$$

Hence

$$u(25) \approx u_2 = 0.084843''$$

5. The radial displacement u of a pressurized hollow thick cylinder (inner radius = 5", outer radius = 8") is given at different radial locations.

| Radius (in) | Radial Displacement (in) |
|-------------|--------------------------|
| 5.0 | 0.0038731 |
| 5.6 | 0.0036165 |
| 6.2 | 0.0034222 |
| 6.8 | 0.0032743 |
| 7.4 | 0.0031618 |
| 8.0 | 0.0030769 |

The maximum normal stress, in psi, on the cylinder is given by

$$\sigma_{\max} = 3.2967 \times 10^6 \left(\frac{u(5)}{5} + 0.3 \frac{du}{dr}(5) \right)$$

The maximum stress, in psi, with second order accuracy is

- (A) 2079.6
- (B) 2104.5
- (C) 2130.7
- (D) 2182.0

Solution

The correct answer is (A).

If we look at the Taylor series

$$u(r + \Delta r) = u(r) + u'(r)\Delta r + \frac{u''(r)(\Delta r)^2}{2} + \frac{u'''(r)(\Delta r)^3}{6} + \dots$$

and

$$u(r + 2\Delta r) = u(r) + u'(r)(2\Delta r) + \frac{u''(r)(2\Delta r)^2}{2} + \frac{u'''(r)(2\Delta r)^3}{6} + \dots$$

Multiply the first equation by 4 and subtract it from the second equation.

$$u(r + 2\Delta r) - 4u(r + \Delta r) = -3u(r) - u'(r)2\Delta r + O(\Delta r)^3$$

$$u'(r)(2\Delta r) = -u(r + 2\Delta r) + 4u(r + \Delta r) - 3u(r) + O(\Delta r)^3$$

$$u'(r) = \frac{-u(r + 2\Delta r) + 4u(r + \Delta r) - 3u(r)}{2\Delta r} + \frac{O(\Delta r)^3}{2\Delta r}$$

$$u'(r) = \frac{-u(r + 2\Delta r) + 4u(r + \Delta r) - 3u(r)}{2\Delta r} + O(\Delta r)^2$$

This equation is second order accurate, that is, the true error is $O(\Delta r)^2$.

$$\frac{du}{dr} \approx \frac{-u(r + 2\Delta r) + 4u(r + \Delta r) - 3u(r)}{2\Delta r}$$

$$r = 5, \Delta r = 0.6$$

$$\begin{aligned}\frac{du}{dr}(5) &\approx \frac{-u(5 + 2 \times 0.6) + 4u(5 + 0.6) - 3u(5)}{2 \times 0.6} \\ &= \frac{-u(6.2) + 4u(5.6) - 3u(5)}{2 \times 0.6} \\ &= \frac{-0.0034222 + 0.014466 - 0.011619}{1.2} \\ &= -0.00047933\end{aligned}$$

Thus,

$$\begin{aligned}\sigma_{\max} &= 3.2967 \times 10^6 \left(\frac{u(5)}{5} + 0.3 \frac{du}{dr}(5) \right) \\ &\approx 3.2967 \times 10^6 \left(\frac{0.0038731}{5} + 0.3 \times (-0.00047933) \right) \\ &= 2079.6 \text{ psi}\end{aligned}$$

6. For a simply supported beam (at $x = 0$ and $x = L$) with a uniform load q , the vertical deflection $v(x)$ is described by the boundary value ordinary differential equation as

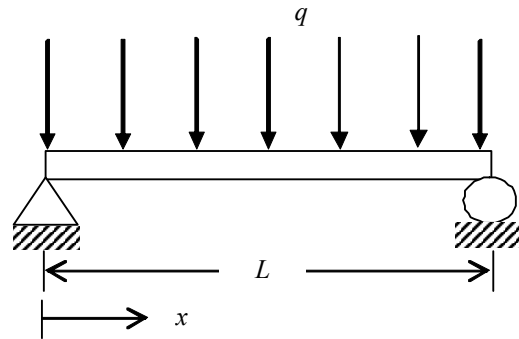
$$\frac{d^2v}{dx^2} = \frac{qx(x-L)}{2EI}, \quad 0 \leq x \leq L$$

where

E = Young's modulus of the beam

I = second moment of area

This ordinary differential equation is based on assuming that $\frac{dv}{dx}$ is small. If $\frac{dv}{dx}$ is not small, then the ordinary differential equation is given by



- (A) $\frac{\frac{d^2v}{dx^2}}{\sqrt{1 + \left(\frac{dv}{dx}\right)^2}} = \frac{qx(x-L)}{2EI}$
- (B) $\frac{\frac{d^2v}{dx^2}}{\left(1 + \left(\frac{dv}{dx}\right)^2\right)^{3/2}} = \frac{qx(x-L)}{2EI}$
- (C) $\frac{\frac{d^2v}{dx^2}}{\sqrt{1 + \left(\frac{dv}{dx}\right)^2}} = \frac{qx(x-L)}{2EI}$
- (D) $\frac{\frac{d^2v}{dx^2}}{1 + \frac{dv}{dx}} = \frac{qx(x-L)}{2EI}$

Solution

The correct answer is (B).

The equation for the deflection in a beam is

$$\frac{1}{\rho} = \frac{M}{EI}$$

Where

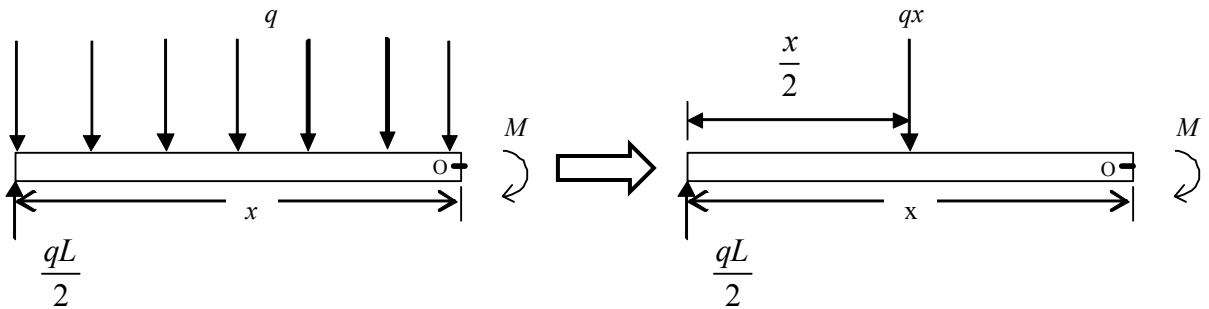
$$\frac{1}{\rho} = \text{curvature of the beam}$$

M = internal moment of the beam where the curvature is to be determined

The curvature of a function v can be rewritten in rectangular format as

$$\frac{1}{\rho} = \frac{\frac{d^2v}{dx^2}}{\left(1 + \left(\frac{dv}{dx}\right)^2\right)^{3/2}}$$

The reaction at each support is $\frac{qL}{2}$.



Writing the balance of the bending moments at point O at a distance x from the left end,

$$M + \frac{qL}{2}x - (qx)\frac{x}{2} = 0$$

$$\begin{aligned} M &= \frac{qx^2}{2} - \frac{qL}{2}x \\ &= \frac{qx(x-L)}{2} \end{aligned}$$

Thus the equation for the deflection of the simply supported beam is

$$\frac{\frac{d^2v}{dx^2}}{\left(1 + \left(\frac{dv}{dx}\right)^2\right)^{3/2}} = \frac{qx(x-L)}{2EI}$$